27°C. A thermod	couple on the oute	er surface of the	sphere indicates
ere is inserted int	o an air stream. A	Assume, and ther	n justify, that the

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Q.3. Attempt any **TWO** parts of the following:

(a) A flat plate of width w = 1 m is maintained at a uniform surface temperature, $T_s = 230^{\circ}$ C, by using independently controlled, electrical strip heaters, each of which is 50 mm long. If atmospheric air at 25°C flows over the plate at a velocity of 60 m/s, at what heater is the

Paper Code: ME 504

B.TECH (SEM V) ODD SEMESTER EXAMINATION 2016-17 HEAT AND MASS TRANSFER

Roll No.

[Time: 3 Hours]

Note: (i) Attempt all questions.

- (ii) Assume missing data suitably, if any.
- (iii) Use appropriate equation / correlation and property data provided.
- (iv) Use of Heisler's Charts is permitted.

Q.1. Attempt any TWO parts of the following:-

- (a) (i) Abot water pipe with outside radius r_1 has a temperature T_1 . A thick insulation applied to reduce the heat loss has an outer radius r_2 and temperature T_2 . On T-r coordinates, sketch the temperature distribution in the insulation for one-dimensional, steady state heat transfer with constant properties. Give brief explanation, justifying the shape of the curve shown.
 - (ii) Derive an expression for critical radius of insulation for cylinder and discuss its significance.
- (b) A copper (k = 401 W/m. K) cable of 30 mm diameter has an electrical resistance of 5×10^{-10} $^{3}\Omega/m$ and is used to carry an electrical current of 250 A. The cable is exposed to an ambient air at 20 °C, and the associated convection coefficient is 25 W/m². K. What are the surface and centerline temperatures of the cable?
- (c) A 10 mm diameter electric wire with a resistance per unit length of $2 \times 10^{-4} \Omega$ / m is coated with an insulation of thermal conductivity k=0.20W/m.K. The insulation is exposed to ambient air at 30 °C for which heat transfer coefficient is10W/m².K. Calculate the value of highest current that may be passed through the wire so that the temperature in any part of insulation does not exceed 200 °C.

Q.2. Attempt any TWO parts of the following:

- (a) The sphere which is 12.7 mm in diameter is at 66° C before it is inserted into an air stream having a temperature of 55°C, 69 s after the sphe sphere behaves as a space-wise isothermal object and calculate the heat transfer coefficient. For pure copper (at 333k) take: ρ =8933 kg/m³, c_{ρ} =389 J//kg K, and k=398 W/m K.
- (b) A plane wall with a thickness of 0.1 m initially at a uniform temperature of 250°C is suddenly immersed in an oil bath at 30 °C. Assuming the convection heat transfer coefficient for the wall in the bath is 500 W/m².K. Assuming one-dimensional heat transfer, calculate the surface temperature of the wall 9 min after immersion. The properties of the wall are k=50 W/m.K, $\rho=7835 \text{ kg/m}^3$, and c=465 J/kg K.
- (c) What do you understand by the "Corrected Length of Fin", how it is beneficial? Two long copper rods (k = 379 W/m. K) of diameter D = 10 mm are soldered together end to end, with solder having a melting point of 650 °C. The rods are in air at 25 °C with a convection coefficient of 10 W/m^2 .K. What is the minimum power input needed to effect the soldering?

[Max. Marks: 100]

(10x2=20)

(10x2=20)

(10x2=20)

electrical input a maximum? What is the value of this input? For Air (at T = 400 K, p = 0.6 atm): $v = 44.02 \times 10^{-6} \text{ m}^2/\text{s}$, k = 0.0338 W/m.K, Pr = 0.690

(b) Explain the phenomenon of free convection on a vertical surface and highlight the role of Grashof Number and Rayleigh number.

<u>OR</u>

Derive the momentum equation for free convection in-terms of volumetric thermal expansion coefficient.

- (c) Answer the following:
 - (i) For laminar/turbulent flow over a flat plate the heat transfer coefficient decreases with the distance from leading edge. Whether the statement is true or false? Substantiate your reply.
 - (ii) Discuss the significance of various dimensionless numbers in forced convection.
- Q.4. Attempt any TWO parts of the following:-
 - (a) A long, thin-walled horizontal tube 100 mm in diameter is maintained at 120°C by the passage of steam through its interior. A radiation shield is installed around the tube, providing an air gap of 10 mm between the tube and the shield, and reaches a surface temperature of 35°C. The tube and the shield are diffuse, gray surfaces with emissivity of 0.8 and 0.1 respectively. What is the radiant heat transfer from the tube per unit length?
 - (b) A flat bottomed hole 6 mm in diameter is drilled to a depth of 24 mm in a diffuse, gray material having an emissivity of 0.8 and a uniform temperature of 1000 K. Determine the radiant power leaving the opening of the cavity (hole). The opening of the cavity may be approximated as a black body at 0 K.
 - (c) What do you understand by the following?
 - (i) Space Resistance
 - (ii) Colored Surface
 - (iii) Plank's Distribution Law
- **Q.5.** Attempt any **TWO** parts of the following:

(10x2=20)

(10x2=20)

- (a) Define Heat Exchanger Effectiveness. Derive an expression for Effectiveness of counter flow heat exchanger in terms of NTU.
 - (b) Derive an expression for molar diffusion rate for steady state equimolar counter diffusion.

<u>OR</u>

Discuss various modes of Pool boiling with the help of Pool boiling curve.

(c) Water (cp =4188 J/kg) at 225 kg/h is to be heated from 35 °C to 95 °C by means of a concentric tube heat exchanger. Oil (cp =2095 J/kg) at 225 kg/h and 210 °C is to be used for heating the water. The overall heat transfer coefficient based on the outer surface of the inner tube is 550 W/m2 K. Determine the length of the exchanger, if the outer diameter of the inner tube is 10 cm.

Correlations:

- 1. For Turbulent flow over a flat plate: $Nu_x = 0.0296 \ Re_x^{4/5} \ Pr1^{/3}$ (0.6 <Pr< 60)
- 2. For Mixed boundary layer flow over a flat plate: $Nu_L = (0.037 \text{ Re}_L^{4/5} - 871) \text{ Pr}1^{1/3}$ (0.6 <Pr< 60; $\text{Re}_{x,c}$ < Re_L < 10⁸; $\text{Re}_{x,c} = 5 \times 10^5$)
- 3. For Double pipe heat exchanger :
 - $\epsilon = 1\text{-} \exp. (-NTU) \qquad ; \qquad \quad \text{for } Cr = 0$
 - $\varepsilon = NTU/(1 + NTU)$; for Cr = 1